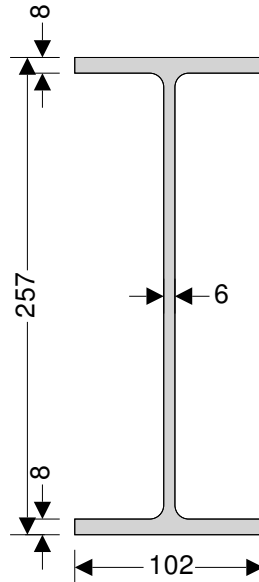


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STEEL BEAM TORSION DESIGN

In accordance with BS5950-1:2000 incorporating Corrigendum No.1

Tedds calculation version 2.0.01



Section details

Section type	UKB 254x102x25
Steel grade	S275
Design strength	$p_{yw} = p_y = 275 \text{ N/mm}^2$
Constant	$\epsilon = \sqrt{(275 \text{ N/mm}^2 / p_y)} = 1.000$

Calculated torsional properties

Shear area of web	$A_{vy} = t \times D = 15.4 \text{ cm}^2$
Plastic modulus of web	$S_{vx} = t \times D^2 / 4 = 99.2 \text{ cm}^3$
Torsion bending constant	$a = \sqrt{(E \times H / (G \times J))} = 966 \text{ mm}$
Normalised warping function	$W_{n0} = (D - T) \times B / 4 = 63.4 \text{ cm}^2$
Statical moment in the flange	$Q_f = (S_x - t \times D^2 / 4) / 4 = 52 \text{ cm}^3$
Statical moment at mid-depth of section	$Q_w = S_x / 2 = 153 \text{ cm}^3$
Warping statical moment	$S_{w1} = (D - T) \times B^2 \times T / 16 = 136 \text{ cm}^4$

Geometry - Beam unrestrained against lateral-torsional buckling between supports.

Effective span	$L = 4000 \text{ mm}$
Length of segment for LT buckling	$L_{LT} = 4000 \text{ mm}$
Compression flanges laterally restrained	
Compression flange partially restrained against rotation on plan	
Effective length for LT buckling	$L_{E,LT} = L_{LT} \times 0.85 = 3400 \text{ mm}$

Loading - Torsional loading comprises only full-length uniformly distributed load(s)

Internal forces & moments on member under factored loading for ult design

Applied shear force	$F_{vy} = 6.0 \text{ kN}$
Maximum bending moment	$M_{LT} = M_x = 11.90 \text{ kNm}$
Applied torque	$T_q = 0.46 \text{ kNm}$



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Minor axis bending moment $M_y = 0$ kNm

Compression force $F_c = 0$ kN

Equivalent uniform moment factors

EUM factor (Cl. 4.3.6.6 and T18) $m_{LT} = 1.000$

Torsional deflection parameters

Beam is torsion fixed and warping free at each end. (as defined in SCI-P-057 section 2.1.6) - Appendix B case 4

Dist along the beam for first derivative of twist $z_1 = 0$ mm

Dist along the beam for second derivative of twist $z_2 = L / 2 = 2000$ mm

First derivative of angle of twist $\phi'_1 = T_q / (G \times J) \times a / L \times [L^2 / (2 \times a) \times (1 / L - 2 \times z_1 / L^2) + \sinh(z_1 / a) - \tanh(L / (2 \times a)) \times \cosh(z_1 / a)] \times 1$ rads = **2.42×10^{-2}** rads/m

Third derivative of angle of twist $\phi'''_1 = T_q / (G \times J \times a^2) \times a / L \times [\sinh(z_1 / a) - \tanh(L / (2 \times a)) \times \cosh(z_1 / a)] \times 1$ rads = **-2.28×10^{-2}** rads/m³

Angle of twist $\phi^2 = T_q \times a / (G \times J) \times a / L \times [L^2 / (2 \times a^2) \times (z_2 / L - z_2^2 / L^2) + \cosh(z_2 / a) - \tanh(L / (2 \times a)) \times \sinh(z_2 / a) - 1] \times 1$ rads = **0.030** rads

Second derivative of angle of twist $\phi''_2 = T_q / (G \times J \times a) \times a / L \times [\cosh(z_2 / a) - \tanh(L / (2 \times a)) \times \sinh(z_2 / a) - 1] \times 1$ rads = **-1.71×10^{-2}** rads/m²

Design parameters

Total angle of twist $\phi = \text{abs}(\phi^2) = 0.030$ rads

First derivative of ϕ $\phi' = \text{abs}(\phi'_1) = 2.42 \times 10^{-2}$ rads/m

Second derivative of ϕ $\phi'' = \text{abs}(\phi''_2) = 1.71 \times 10^{-2}$ rads/m²

Third derivative of ϕ $\phi''' = \text{abs}(\phi'''_1) = 2.28 \times 10^{-2}$ rads/m³

Section classification

$b / T = 6.1$

$d / t = 37.5$

$r_{1s} = \min(1.0, \max(-1.0, F_c / (d \times t \times p_{yw}))) = 0.000$

$r_{2s} = F_c / (A_g \times p_{yw}) = 0.000$

Section classification is plastic

Shear capacity (parallel to y-axis)

Design shear force $F_{vy} = 6.0$ kN

Design shear resistance (Cl. 4.2.3) $P_{vy} = 0.6 \times p_y \times A_{vy} = 254.6$ kN

Pass - Shear

Moment capacity (x-axis)

Design bending moment $M_x = 11.9$ kNm

Moment capacity $M_{cxu} = p_y \times S_x = 84.0$ kNm

Moment capacity low shear (Cl. 4.2.5.1) $M_{cx} = \min(p_y \times S_x, 1.2 \times p_y \times Z_x) = 84.0$ kNm

Pass - Moment capacity exceeds design bending moment

Lateral torsional buckling

Effective length for lateral torsional buckling $L_{E_LT} = 3400$ mm

Slenderness ratio $\lambda = L_{E_LT} / r_y = 158$

Buckling parameter $u = 0.866$

Flange ratio $\eta = 0.5$

Torsional index $x = 31.5$

Slenderness factor $v = 1 / (1 + 0.05 \times (\lambda / x)^2)^{0.25} = 0.82$

Ratio - cl 4.3.6.9 $\beta_w = 1.0 = 1.000$



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Equivalent slenderness – cl 4.3.6.7

$$\lambda_{LT} = u \times v \times \lambda \times \sqrt{(\beta_w)} = \mathbf{111}$$

Limiting slenderness – Annex B2.2

$$\lambda_{L0} = 0.4 \times \sqrt{(\pi^2 \times E_{S5950} / p_y)} = \mathbf{34}$$

Euler stress

$$p_E = \pi^2 \times E_{S5950} / \lambda_{LT}^2 = \mathbf{163 \text{ N/mm}^2}$$

Perry factor

$$\eta_{LT} = \max(7.0 \times (\lambda_{LT} - \lambda_{L0}) / 1000, 0) = \mathbf{0.540}$$

$$\phi_{LT} = (p_y + (\eta_{LT} + 1) \times p_E) / 2 = \mathbf{262880916.767}$$

Bending strength

$$p_b = p_E \times p_y / (\phi_{LT} + \sqrt{(\phi_{LT}^2 - p_E \times p_y)}) = \mathbf{107 \text{ N/mm}^2}$$

Buckling resistance moment

$$M_b = p_b \times S_x = \mathbf{32.7 \text{ kNm}}$$

Max moment governing buckling resistance

$$M_{LT} = \mathbf{11.9 \text{ kNm}}$$

Equiv uniform moment factor for LTB

$$m_{LT} = \mathbf{1.00}$$

$$M_b / m_{LT} = \mathbf{32.7 \text{ kNm}}$$

Pass - lat. tors. buckling

Buckling under combined bending & torsion -SCI-P-057 section 2.3

For simplicity, a conservative check is applied using the maximum stresses due to each of the separate load effects, even though these do not necessarily all occur at the same section along the member.

Span factor

$$L / a = \mathbf{4.14}$$

Angle of twist

$$\phi = \mathbf{0.030 \text{ rads}}$$

Second derivative of ϕ

$$\phi'' = \mathbf{17.1 \times 10^{-3} \text{ rads/m}^2}$$

Induced minor axis moment

$$M_{yt} = M_x \times \phi / 1 \text{ rad} = \mathbf{0.35 \text{ kNm}}$$

Normal stress at flange tip due to M_{yt}

$$\sigma_{yt} = M_{yt} / Z_y = \mathbf{12 \text{ N/mm}^2}$$

Normal stress at flange tip due to warping

$$\sigma_w = E_{S5950} \times W_{n0} \times \phi'' / 1 \text{ rad} = \mathbf{22 \text{ N/mm}^2}$$

Interaction index

$$i_b = M_x \times m_{LT} / M_b + (\sigma_{yt} + \sigma_w) / p_y \times (1 + 0.5 \times M_x \times m_{LT} / M_b) = \mathbf{0.51}$$

Pass - Combined bending and torsion check satisfied

Local capacity under combined bending & torsion

For simplicity, a conservative check is applied using the maximum stresses due to each of the separate load effects, even though these do not necessarily all occur at the same section along the member.

Max. direct stress due to M_x

$$\sigma_x = M_x / Z_x = \mathbf{45 \text{ N/mm}^2}$$

Combined stress - eqn 2.22

$$\sigma_x + \sigma_{yt} + \sigma_w = \mathbf{79 \text{ N/mm}^2}$$

Design strength

$$p_y = \mathbf{275 \text{ N/mm}^2}$$

Pass - Local capacity

Combined shear stresses - SCI-P-057 section 2.3

For simplicity, a conservative check is applied using the maximum shear stresses due to each of the separate load effects, even though these do not necessarily all occur at the same section along the member.

Max shear stresses due to bending in web

$$\tau_{bw} = F_{vy} \times Q_w / (I_x \times t) = \mathbf{4 \text{ N/mm}^2}$$

Max shear stresses due to bending in flange

$$\tau_{bf} = F_{vy} \times Q_f / (I_x \times T) = \mathbf{1 \text{ N/mm}^2}$$

Max shear stresses due to torsion in web

$$\tau_{tw} = \text{abs}(G \times t \times \phi' / 1 \text{ rad}) = \mathbf{11 \text{ N/mm}^2}$$

Max shear stresses due to torsion in flange

$$\tau_{tf} = \text{abs}(G \times T \times \phi' / 1 \text{ rad}) = \mathbf{16 \text{ N/mm}^2}$$

Max shear stresses due to warping in flange

$$\tau_{wf} = \text{abs}(-E_{S5950} \times S_{w1} \times \phi''' / 1 \text{ rad} / T) = \mathbf{1 \text{ N/mm}^2}$$

Amp shear stress torsion & warping in web

$$\tau_{vtw} = \tau_{tw} \times (1 + 0.5 \times M_x \times m_{LT} / M_b) = \mathbf{14 \text{ N/mm}^2}$$

Amp shear stress torsion & warping in flange

$$\tau_{vtf} = (\tau_{tf} + \tau_{wf}) \times (1 + 0.5 \times M_x \times m_{LT} / M_b) = \mathbf{20 \text{ N/mm}^2}$$

Combined shear stresses due to bending, torsion & warping:

Combined shear stresses in web

$$\tau_w = \tau_{bw} + \tau_{vtw} = \mathbf{18 \text{ N/mm}^2}$$

Combined shear stresses in flange

$$\tau_f = \tau_{bf} + \tau_{vtf} = \mathbf{21 \text{ N/mm}^2}$$

Shear strength

$$p_v = 0.6 \times p_y = \mathbf{165 \text{ N/mm}^2}$$

Pass - Combined shear stresses



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Twist check

Total applied torque (unfactored)

$$T_{qu} = 0.46 \text{ kNm}$$

Maximum twist under sls loading

$$\phi_{sls} = \phi \times T_{qu} / T_q = 1.69 \text{ deg}$$

Twist limit

$$\phi_{lim} = 2.00 \text{ deg}$$

Pass - Twist